

Objects that are flexible purely for geometric reasons (sheets, filaments and ribbons) make an overwhelming variety of patterns in nature and our technological world.

Can we organize this profusion of shape and form by identifying building blocks? Are there elementary excitations of elastic materials that we can study?



Sea urchin



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Sharon, Swinney, Marder

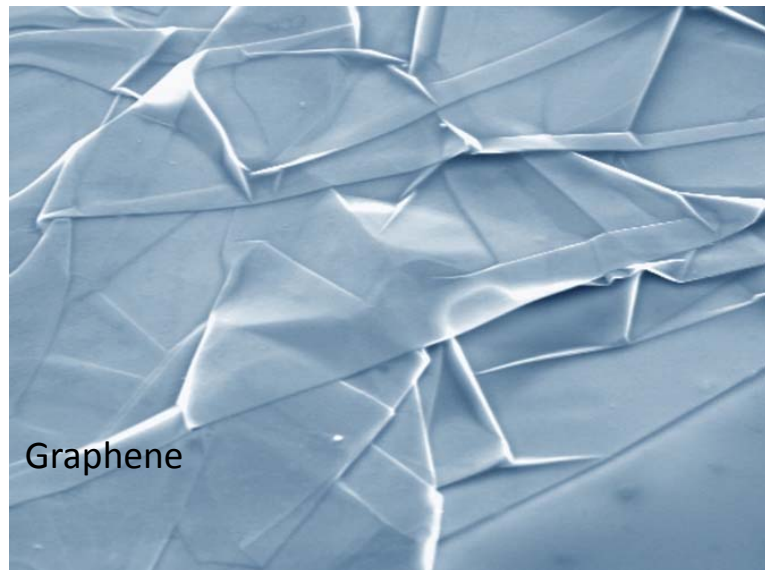


Yva Momatiuk and John Eastcott/PhotoResearchers, Inc.

that a leaf or flower—just like a torn sheet of plastic—can use an enhanced, uniform growth at its margins to generate such complex patterns. Examples of wavy edges in nature include, from left to right, some lichens (shown, *Sticta limbata*), orchids (shown, *Schomborgkia beysiana*), sea slugs (represented by *Glossodoris hikuensis*) and ornamental cabbage. (Lichen photograph courtesy of Stephen Sharnoff; sea slug photograph courtesy of Jeff Jeffords.)



abric

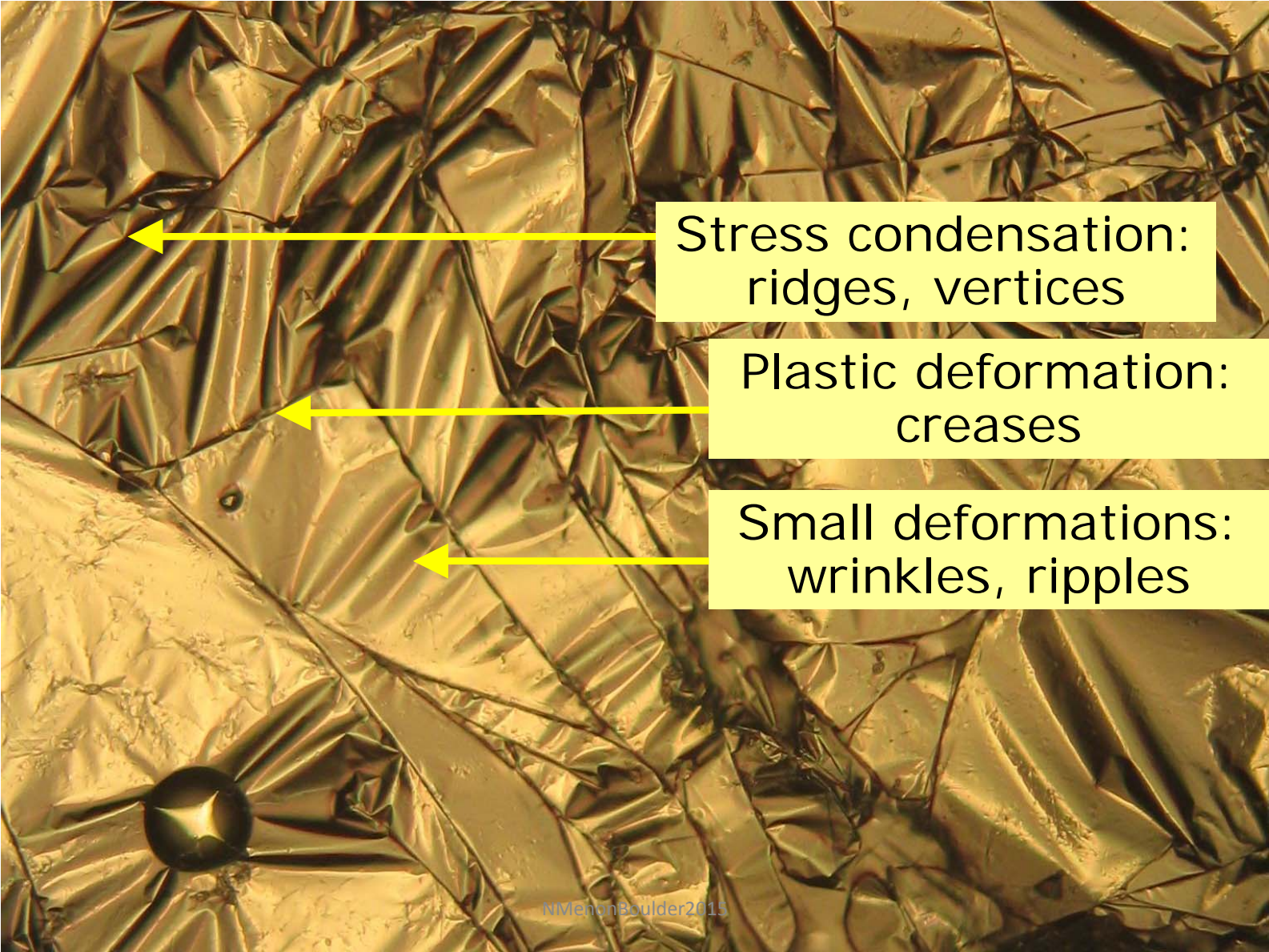


Graphene



Earth's skin

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Stress condensation:
ridges, vertices

Plastic deformation:
creases

Small deformations:
wrinkles, ripples

Overall goals of our discussion



- These structures are generated by elastic instabilities
- What are the energetics and stability of these constructs?
- Where do all these structures belong?
- How to specify these axes?

Material property

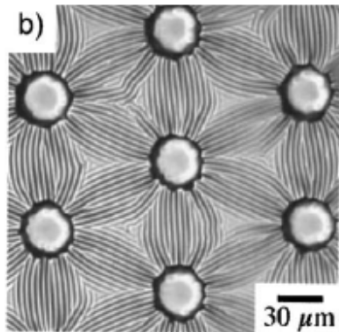
??

External forces or confinement or growth (structureless)

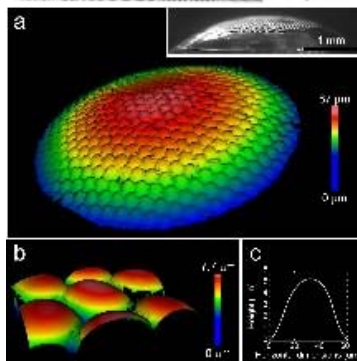
Instability not as “failure” but technological tool

Patterning (actuatable ones at that), metrology, coatings, surface control

Nanoscale elastic patterning

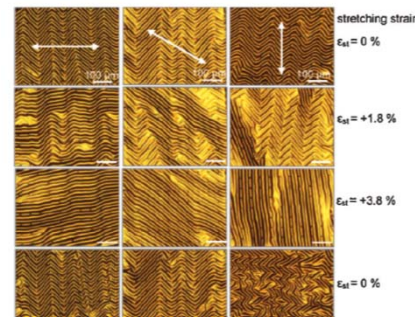


Bowden et al, 1999

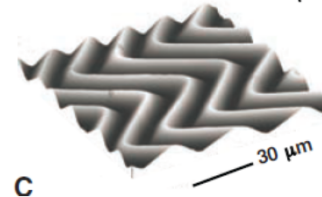
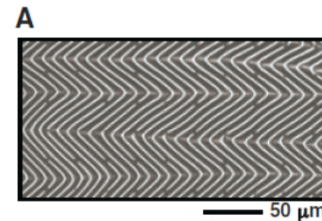


Crosby, Breid 2010

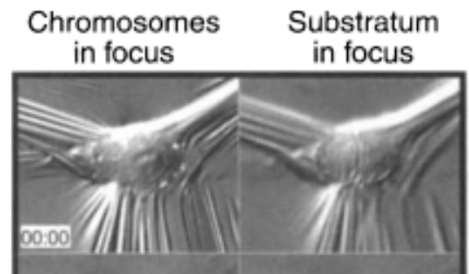
Stretchable electronics



Rogers 2011



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Burton, Nature 97

Plan

Overall theme	Pattern formation via elastic instabilities
Elasticity	Stress, large deformation strain, Hooke's Law; moduli for plates
1D Euler buckling	two approaches
1D wrinkling mode (briefly)	Scaling analysis, generality of "substrate"; going beyond single
1D Folds	mechanical stability, exact solution, system size dependence
2D Wrinkling dimensionality of phase space; (briefly) other geometries	Lamé problem as archetype, two limits of FvK, increased
Crumples	Ridges, d-cones and e-cones;
Wrapping	Idea of asymptotic isometry; Folds in 2-D

Things I will not do

Mainly mechanics, will not work at thermal scales

Focus on sheets, not on filaments, ribbons (but others have more than compensated for that)

No free surface instabilities

Focus on statics, not on dynamics (lots of open problems and opportunities here)

Strain describes change in distance between material points

Material point $\vec{x} \rightarrow \vec{x}'$ moves from

get a displacement field

$$u_i = x'_i - x_i$$

Change in distance between material points $d\vec{x}$ apart

$$\begin{aligned} dl'^2 &= dx_i dx_i \\ dl'^2 &= dx'_i dx'_i = (dx_i + du_i)(dx_i + du_i) \\ &= (dx_i + \partial_j u_i dx_j)(dx_i + \partial_j u_i dx_j) \\ &= dl^2 + 2 \partial_j u_i dx_i dx_j + \partial_i u_i \partial_k u_k dx_i dx_k \\ &= dl^2 + 2 \epsilon_{ij} dx_i dx_j \end{aligned}$$

Strain tensor

$$\epsilon_{ij} = \frac{1}{2} (\partial_i u_j + \partial_j u_i + \partial_i u_k \partial_j u_k)$$

Sometimes called

Nonlinear in deformation.

Green's strain tensor (as opposed to linear Cauchy)

Stress-Strain Relation

— Hooke's Law

Young's modulus $Y = \frac{\sigma_{xx}}{\epsilon_{xx}}$

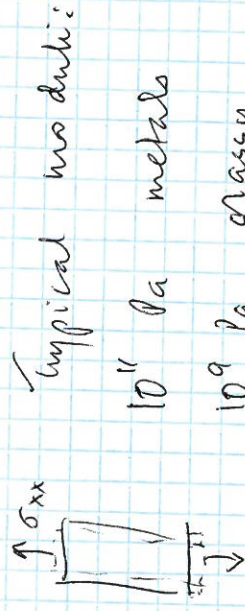
Poisson ratio $\lambda = -\frac{\epsilon_{yy}}{\epsilon_{xx}}$

$\sigma_{ij} = 2\mu \epsilon_{ij} + \lambda \delta_{ij} \epsilon_{kk}$

(Cauchy 1822, Lamé 1852)

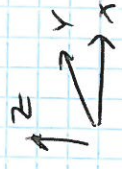
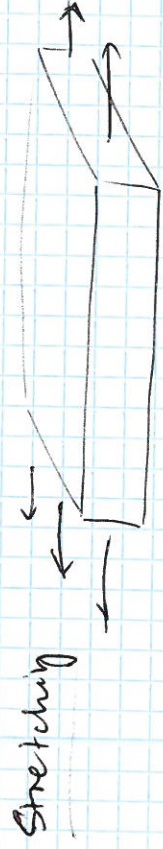
OR $\epsilon_{ij} = \frac{(1+\lambda)}{E} \sigma_{ij} - \frac{\lambda}{E} \sigma_{kk} \delta_{ij}$

Explicitly $\epsilon_{xx} = \frac{\sigma_{xx}}{E} - \frac{\lambda}{E} (\sigma_{yy} + \sigma_{zz})$; $\epsilon_{xy} = \frac{1+\lambda}{E} \sigma_{xy}$



typical moduli:
 10^{11} Pa metals
 10^9 Pa glassy polymers
 $10^8 - 10^9$ Pa elastomers

Thin plates: Not interested in variations across thickness. Integrate out z-dependence



$$\epsilon_{xx} = \frac{\sigma_{xx}}{E} = F \left(\frac{Et}{E} \right)$$

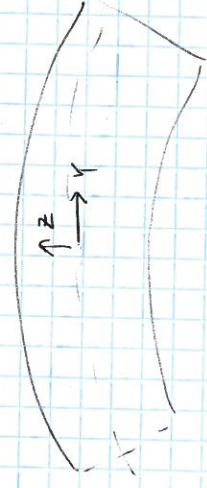
$Y = Et$ ← stretching modulus.

Stretching energy (area) $E = \frac{1}{2} Y \epsilon_{xx}^2$

$$Y = Et$$

Bending

[Also look at Alex Levine notes]



Deflection $w = w(y)$

$$\epsilon_{yy} = \frac{\partial^2 w}{\partial y^2}$$

$$\sigma_{yy} = \frac{\sigma_{yy}}{E} = A \frac{\sigma_{yy}}{E}$$

$$\epsilon_{xx} = \frac{\sigma_{xx}}{E} = A \frac{\sigma_{yy}}{E} = 0$$

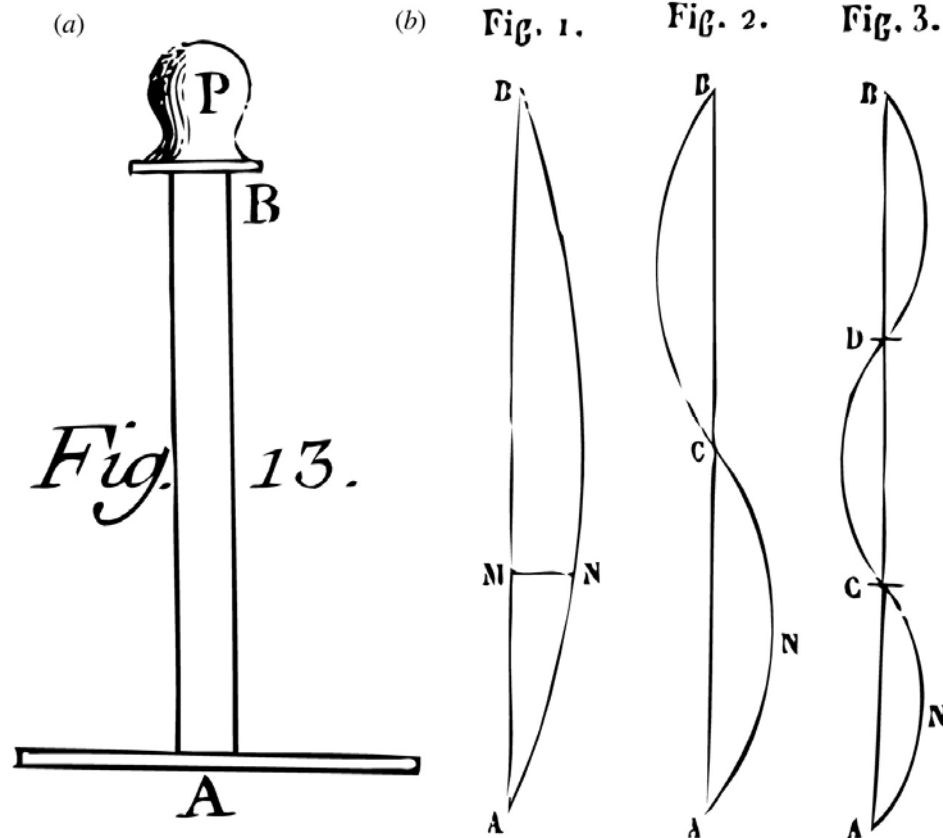
$$\Rightarrow \sigma_{xx} = A \sigma_{yy}$$

$$\epsilon_{yy} = \frac{1-\nu^2}{E} \cdot \sigma_{yy} \Rightarrow \sigma_{yy} = \frac{E \epsilon_{yy}}{1-\nu^2} = -\frac{Ez}{1-\nu^2} \frac{\partial^2 w}{\partial y^2}$$

Bending moment $M_y =$

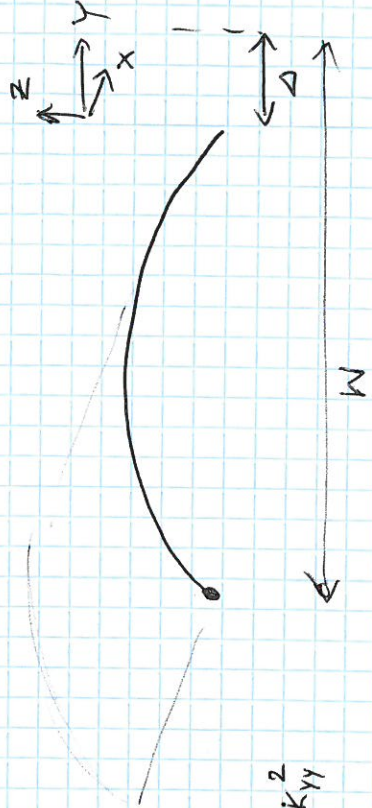
$$\int_{-t/2}^{t/2} \sigma_{yy} z dz = - \int_{-t/2}^{t/2} \frac{Ez^2}{1-\nu^2} \frac{\partial^2 w}{\partial y^2} dz = B \frac{\partial^2 w}{\partial y^2} \rightarrow B = \frac{Et^3}{12(1-\nu^2)}$$

Euler buckling (a) illustrations from Euler (1744) (b) illustrations from Lagrange 1770.



Alain Goriely et al. Proc. R. Soc. A 2008;464:3003-3019

Euler Buckling of a plate



Elastic energy density

$$e = \frac{1}{2} Y \epsilon_{yy}^2 + \frac{1}{2} B \kappa_{yy}^2$$

We analyze this situation in two ways

① Elastica

Compare pure compression against pure bending.

- Compression $\epsilon_{yy} = \frac{A}{W} \Rightarrow e_s = \frac{1}{2} Y \left(\frac{A}{W}\right)^2$

- Bending - assume inextensibility constraint

$$\int_0^{W-\Delta} dy \sqrt{1 + \xi'^2} - W = 0$$

Small slope $|\xi'| \ll 1 \rightarrow \int_0^{W-\Delta} (1 + \frac{1}{2} \xi'^2) dy - W = 0$

$$e = \frac{1}{2} B \xi''^2 + \sigma \left(\frac{1}{2} \xi'^2 - A \right) = 0$$

Euler Lagrange: $B \xi'''' - \sigma \xi'' = 0$

With torque = 0 & $\xi(0) = \xi(W) = 0$

$$\xi(y) = A \sin\left(\sqrt{\frac{\sigma}{B}} y\right)$$

$$A = \frac{2}{\pi} \sqrt{\Delta W}$$

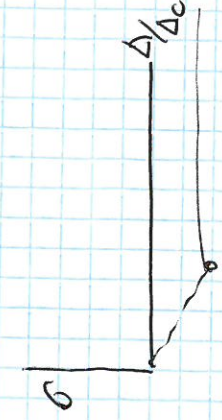
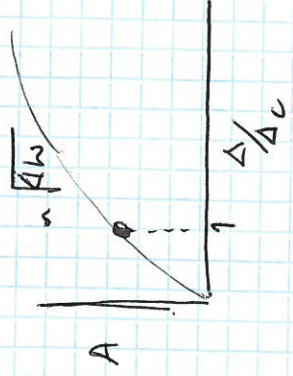
$$e = \frac{1}{2} \frac{B A}{W^3}$$

Transition at $\Delta/\Delta_c = 1$ where

$$\frac{B \Delta_c}{W^3} = \frac{Y \Delta_c^2}{W^2}$$

$$\Rightarrow \Delta_c = \frac{B}{Y W}$$

$$= \frac{B}{Y W^2} \sqrt{\frac{t}{W}}$$



② Full treatment of Euler buckling in small slope

$$\vec{u}(y) = u(y)\hat{y} + \zeta(y)\hat{z}$$

$$E_{yy} = u' + \frac{1}{2}\zeta'^2 \quad (\text{omitting a } \frac{1}{2}u'^2 \text{ term})$$

$$\sigma_{yy} = \gamma E_{yy} \quad \text{Energy density, } \rho = \frac{1}{2}\gamma E_{yy}^2 + \frac{1}{2}B(\zeta'')^2$$

The corresponding Euler-Lagrange equations are

$$\textcircled{1} B\zeta'''' - \sigma_{yy}\zeta'' = 0 \quad \text{- Normal force balance, from variation w.r.t. } \zeta(y)$$

$$\textcircled{2} \sigma_{yy}' = 0 \Rightarrow \sigma_{yy} = \text{const.} - \text{In-plane force balance from variation w.r.t. } u(y)$$

This pair of Föppl-von Kármán equations may be solved by a series solution.

$$\text{Try } \zeta(y) \approx A\left(\frac{\pi}{W}\right) \sin\left(\frac{\pi y}{W}\right) + \text{other terms}$$

$$u(y) \approx -\frac{\Delta}{W} + \text{other terms}$$

Plug $\zeta(y)$ into the first FvK equation.

$$B\left(\frac{\pi}{W}\right)^4 A - \sigma_{yy}\left(\frac{\pi}{W}\right)^2 A = 0 \Rightarrow \sigma_{yy} = \frac{B\pi^2}{W^2} \quad \textcircled{3}$$

$$\text{But } \sigma_{yy} = \gamma E_{yy} = \gamma \left[-\frac{\Delta}{W} + \frac{1}{2} \frac{\pi^2}{W^2} \sin^2\left(\frac{\pi y}{W}\right) \right]$$

$$= \gamma \left[-\frac{\Delta}{W} + \frac{\pi^2}{W^2} \frac{A^2}{4} + \text{oscillating term which will have} \right] \quad \text{to be fixed by other terms in series} \quad \textcircled{4}$$

Put ③ & ④ together, to get

$$A^2 = \frac{Z^2}{K^2} W (A - A_c) \quad \text{where} \quad \frac{A_c}{W} = \frac{K^2 B}{W^2 Y}$$

The first term in this attempt at a series solution (we won't talk about convergence) already yields the same threshold as in the elastica i.e. A_c/W .

Further $A \sim \sqrt{W} \sqrt{A - A_c}$ goes to the elastica solution as $A \gg A_c$

We were lucky to have a solution in this single problem, but both in this problem, and in other instabilities to come, can do linear stability analysis

for small $\frac{A - A_c}{A_c}$: called post-buckling.

Here for large Δ

$\frac{A}{A_c} \gg 1$ get back elastica.

Wrinkles in 1D

Cerda and Mahadevan PRL 2003

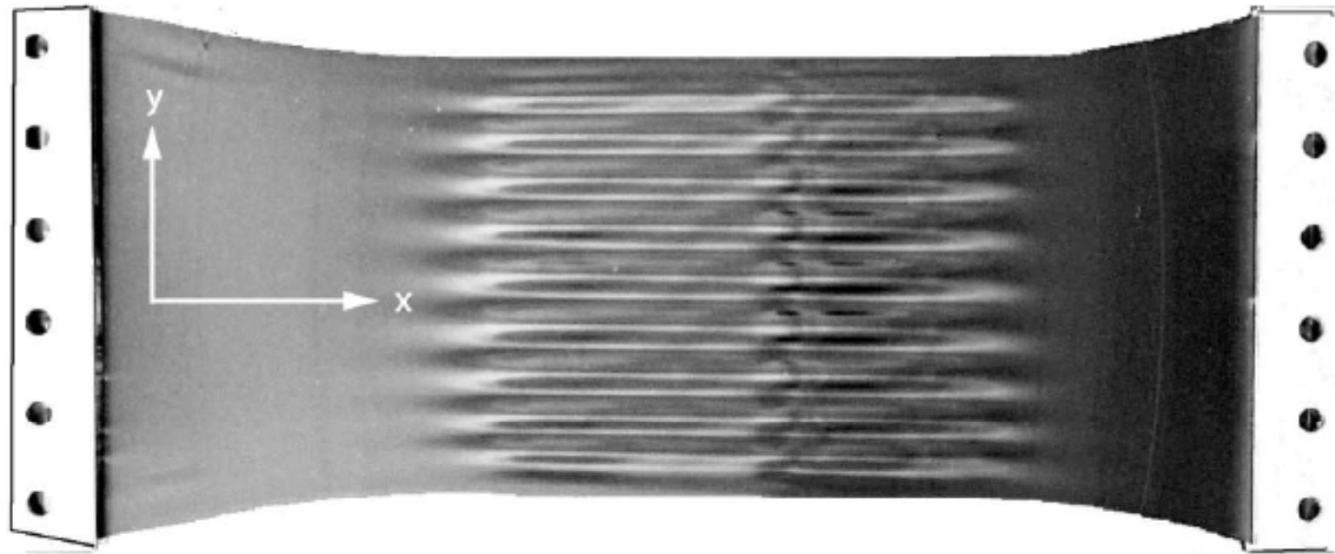
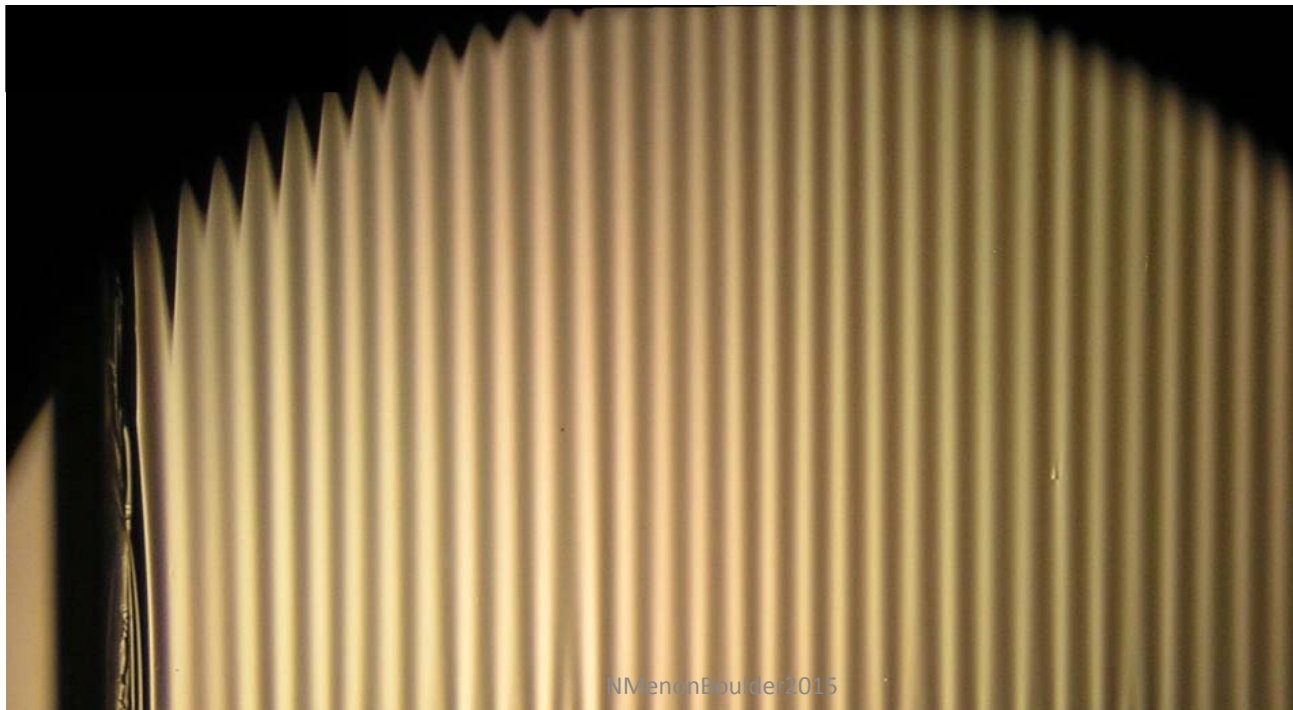


FIG. 1. Wrinkles in a polyethylene sheet of length $L \approx 25$ cm, width $W \approx 10$ cm, and thickness $t \approx 0.01$ cm under a uniaxial tensile strain $\gamma \approx 0.10$. (Figure courtesy of K. Ravi-Chandar)

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Wrinkles in 1D



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Huang PRL 2010

1) Winkler a la Cerda - Mahadevan

Start with the case of a fluid foundation.

$$\text{Energy density } e = \frac{1}{2} B \left(\frac{\partial \zeta}{\partial x} \right)^2 + \frac{1}{2} K \zeta^2$$

Inextensible

$$\frac{\Delta l}{l} = -\frac{\Delta}{W} = \frac{\lambda}{l}$$

$$l = \int_0^{\lambda} \sqrt{1 + \left(\frac{\partial \zeta}{\partial x} \right)^2} dx$$

$$\approx \int_0^{\lambda} \left(1 + \frac{1}{2} \left(\frac{\partial \zeta}{\partial x} \right)^2 \right) dx = \lambda + \frac{1}{4} q^2 A^2$$

$$-\frac{q^2 A^2}{4} = -\frac{\Delta}{W} \Rightarrow qA = 2 \sqrt{\frac{\Delta}{W}}$$

Put this in the energy density.

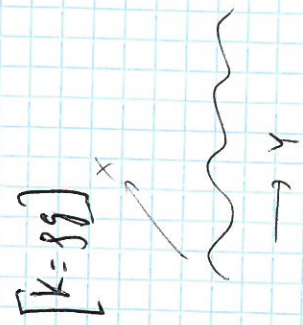
$$e = \frac{1}{2} \left[B (q^2 A)^2 + K A^2 \right] = \frac{1}{2} \left[B q^4 A^2 + K A^2 \right]$$

$$= \frac{1}{2} \left[B q^2 \left(\frac{4\Delta}{W} \right) + K \left(\frac{4\Delta}{W} \right) \frac{1}{q^2} \right]$$

Minimize to get $q \propto \left(\frac{K}{B} \right)^{1/4}$

Tensional case $\frac{T}{2} \left(\frac{\partial \zeta}{\partial x} \right)^2$

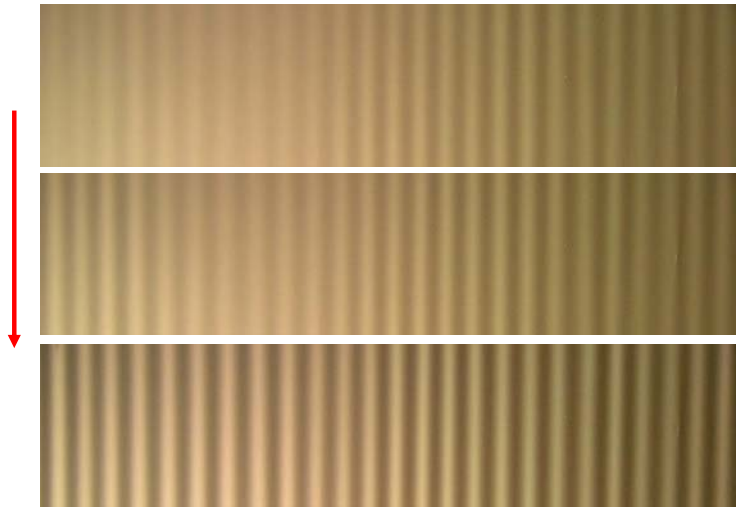
$$\text{Here } K = \frac{T}{L^2} ; qA = 2 \sqrt{\delta \Delta}$$



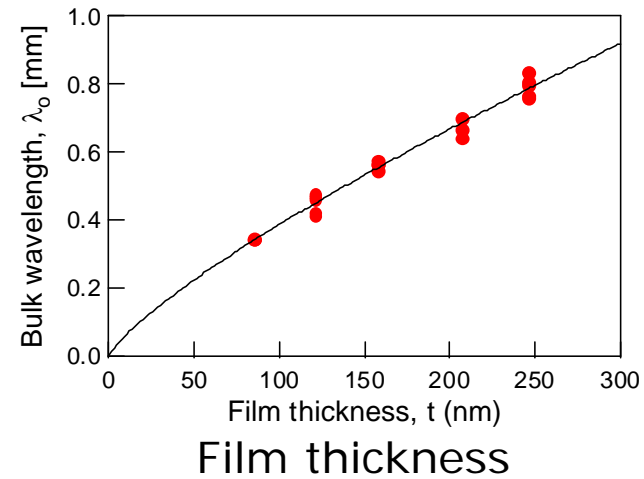
$$\zeta = A \sin(qx)$$

(Please find threshold)

Wrinkles in 1D – fluid substrate



t=246 nm, increasing compression



Wavelength independent of amplitude

$$q_0^{-1} = \lambda_0 = \left(\frac{B}{\rho g}\right)^{1/4}$$

Tuning wavelength through B

Thickness

Young's modulus

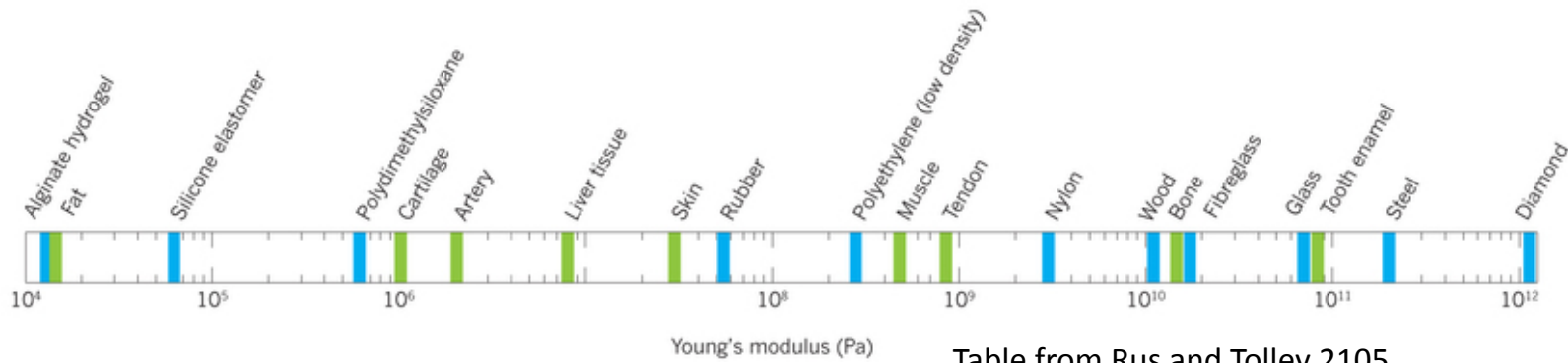
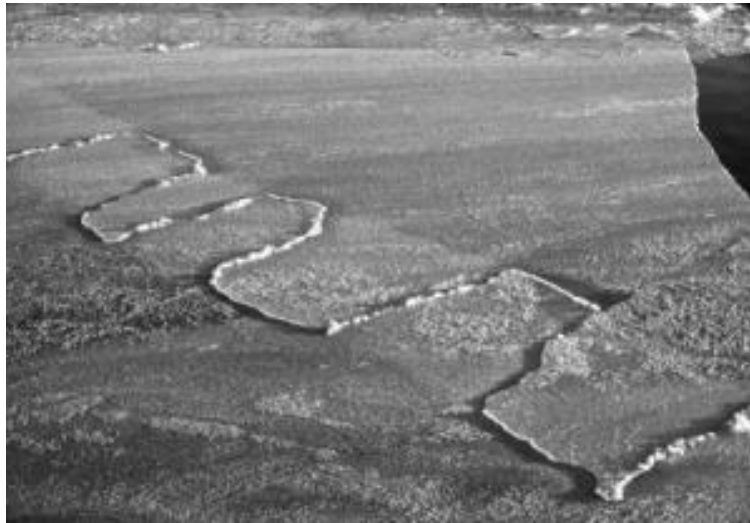


Table from Rus and Tolley 2105

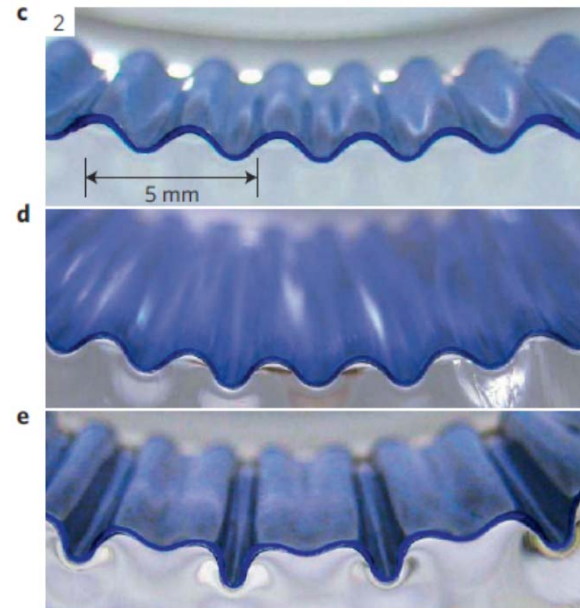
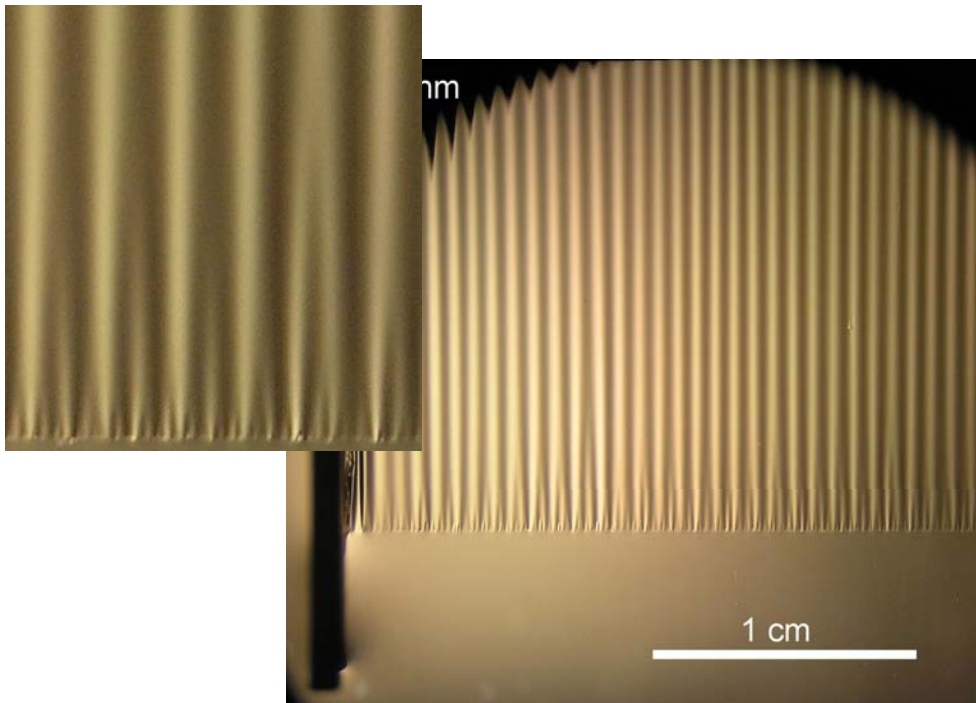
Finger rafting

Vella and Wettlaufer, PRL 2004



Finger rafting is the block zippered pattern that forms when thin ice sheets floating on water collide creating "fingers" that push over and under each other alternately. This photo was taken off the Antarctic coast. (Credit: W.F. Weeks)

Wrinkles in 1D – beyond single mode



Period doubling phenomena Brau et al 2010

Cascade between two wavelengths, Huang 2010

Lecture 1:

Main source for wrinkling calculation -

Cerda, E., & Mahadevan, L. (2003). Geometry and physics of wrinkling. *Physical review letters*, 90(7), 074302.

Discussion of Euler buckling regimes follows a pedagogical review in preparation by Benny Davidovitch and myself. Get in touch with me if you want a draft when it is ready (end of summer 2015?)

I have cited data and images where I showed them.

Useful (to me) books on elasticity:

Physics of Continuum Matter by B. Lautrup -- *nice exposition at an introductory level*

Elasticity by Landau and Lifshitz – *no comments needed*

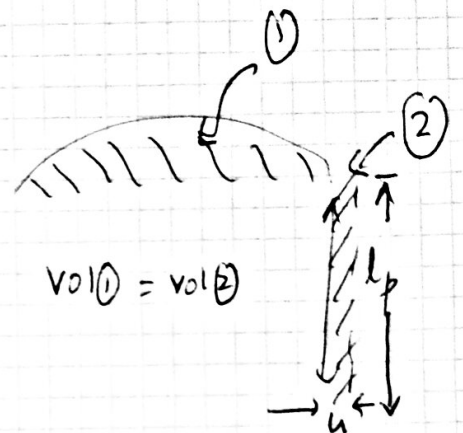
Theory of Elasticity by Timoshenko and Goodier; Plates and Shells by Timoshenko and Woinowsky-Krieger – *both books are detailed expositions by major figure in engineering mechanics, good place to look up solutions for specific geometries*

Soft substrate

Incompressible substrate

Typical displacement of u , over a penetration depth $l_p \Rightarrow A\lambda \sim ul_p$

Typical strain $\epsilon \sim \frac{u}{l_p} \sim \frac{A\lambda}{l_p^2}$



$$\text{Energy/area} \sim \frac{E_s \epsilon^2 (l_p \lambda)}{\lambda} \sim E_s \left(\frac{A\lambda}{l_p^2} \right)^2 l_p$$

$$\sim \left[\frac{E_s A^2}{l_p^3} \right] A^2$$

$$\text{For deep substrate } l_p \sim \lambda \quad K \sim \frac{E_s}{A}$$

$$\text{For shallow substrate } l_p \sim H \quad K \sim \frac{E_s \lambda^2}{H^3}$$

where H is the depth of the substrate.